

# GLOBAL STABILITY AND SENSITIVITY ANALYSIS OF NUMERICAL AND EXPERIMENTAL FLOW FIELDS WITH FOCUS ON FLOW CONTROL

Simone Camarri

University of Pisa

### Contributing authors



- Dr. Andrea Fani, University of Pisa, now at EPFL Lausanne
- <u>Dr. B. Fallenius</u>, KTH Stockholm
- Prof. Jens Fransson, KTH Stockholm
- Dr. Flavio Giannetti, University of Salerno
- <u>Prof. Angelo Iollo</u>, University of Bordeaux
- <u>Prof. Paolo Luchini</u>, University of Salerno
- <u>Prof. Maria Vittoria Salvetti,</u> University of Pisa

#### Introduction



• Global stability analysis provides important information for several classes of unstable flows

• Adjoint methods are often used to study the sensitivity of a global instability to a wide range of perturbations

• provide information on the nature and hidden characteristics of the instability

- provide hints on how to control the investigated instabilities
- Rigorous application of these methods is limited to low Reynolds numbers

• For given classes of flows (bluff-bodies), application of these methods to mean flow fields, even neglecting Reynolds stresses, provide accurate estimation of some properties of the saturated instability

### Overview



- Global stability and sensitivity analysis: a concise introduction
- Applications with focus on:
  - Flow control
  - Application to a case at high Reynolds number



### Global stability and sensitivity analysis: a concise introduction



### Global stability analysis

• Starting point: a steady solution of the NS equations (Baseflow Ub):

$$\mathbf{U}_b \cdot \nabla \mathbf{U}_b + \nabla P_b - \frac{1}{Re} \nabla^2 \mathbf{U}_b = \mathbf{0}$$
$$\nabla \cdot \mathbf{U}_b = 0$$

• Perturbation of Ub in modal form:

$$\mathbf{U}(\mathbf{x},t) = \mathbf{U}_b(\mathbf{x},t) + \epsilon \,\mathbf{u}(\mathbf{x}) \,\mathrm{e}^{\sigma \,t}$$
$$P(\mathbf{x},t) = P_b(\mathbf{x},t) + \epsilon \,p(\mathbf{x}) \,\mathrm{e}^{\sigma \,t}$$

• Linearized ( $\epsilon <<1$ ) dynamics of the perturbation: resulting eigenvalue problem:

$$\sigma \mathbf{u} + \mathbf{u} \cdot \nabla \mathbf{U}_b + \mathbf{U}_b \cdot \nabla \mathbf{u} + \nabla p - \frac{1}{Re} \nabla^2 \mathbf{u} = \mathbf{0}$$
$$\nabla \cdot \mathbf{u} = 0$$

- Given the mode  $(\sigma, \mathbf{u}, p)$ 

 $\operatorname{Real}(\sigma)$  amplification factor (>0 unstable)

 $\operatorname{Imag}(\sigma)/(2\pi)$  frequency of the mode

### Sensitivity analysis



- > Stability properties can be affected by modifications of the flow
- Sensitivity of the instability to particular modifications add information on its physical origin/properties
- Information on sensitivity can be used also to control the instability by proper modifications on the flow

Adjoint methods have been used in the last years to systematically characterize the sensitivity of unstable modes to various parameters in the linear framework



### Sensitivity analysis

• When the eigenvalue problem is perturbed each eigenfunction/ eigenvalue changes consequently:



- $\bullet$  Objective: study the variation of a considered eigenvalue  $\sigma$
- Linear framework: the analysis is linearized

### Sensitivity analysis



- Different perturbations can be applied. For instance:
  - 1) perturbations acting only on the stability equations
  - 2) Generic perturbations of the base flow field
  - 3) Perturbations acting on the baseflow equations





perturbation of the linearized perturbation equations

Sensitivity analysis

• Unperturbed problem:

$$\sigma \mathbf{u} + \mathbf{u} \cdot \nabla \mathbf{U}_b + \mathbf{U}_b \cdot \nabla \mathbf{u} + \nabla p - \frac{1}{Re} \nabla^2 \mathbf{u} = \mathbf{0}$$
$$\nabla \cdot \mathbf{u} = 0$$





perturbation of the linearized perturbation equations

Sensitivity analysis

• Perturbed problem ( $\delta H$  linear):

$$\tilde{\sigma}\mathbf{u} + \tilde{\mathbf{u}} \cdot \nabla \mathbf{U}_b + \mathbf{U}_b \cdot \nabla \tilde{\mathbf{u}} + \tilde{\nabla p} - \frac{1}{Re} \nabla^2 \tilde{\mathbf{u}} = \delta \mathbf{H}(\mathbf{u}, p)$$

$$\nabla \cdot \tilde{\mathbf{u}} = 0$$
Perturbation
(linear function)





perturbation of the linearized perturbation equations

Sensitivity analysis

• Perturbed problem (<u>δ**H** linear</u>):

$$\begin{split} \tilde{\sigma}\mathbf{u} + \tilde{\mathbf{u}} \cdot \nabla \mathbf{U}_b + \mathbf{U}_b \cdot \nabla \tilde{\mathbf{u}} + \tilde{\nabla p} - \frac{1}{Re} \nabla^2 \tilde{\mathbf{u}} = \delta \mathbf{H}(\mathbf{u}, p) \\ \nabla \cdot \tilde{\mathbf{u}} = 0 \end{split} \tag{Perturbation}$$

• Result of the sensitivity analysis for a mode  $(\sigma, \mathbf{u}, p)$ 

Perturbation (linear functional)

$$\delta \sigma = \frac{\langle \mathbf{u}^+, \delta \mathbf{H}(\mathbf{u}, \mathbf{p}) \rangle}{\langle \mathbf{u}^+, \mathbf{u} \rangle}$$

Adjoint eigenvalue problem associated to the linearized equations

$$\sigma^* \mathbf{u}^+ + \nabla \mathbf{U}_b \cdot \mathbf{u}^+ - \mathbf{U}_b \cdot \nabla \mathbf{u}^+ + \nabla p^+ - \frac{1}{Re} \nabla^2 \mathbf{u}^+ = \mathbf{0}$$
$$\nabla \cdot \mathbf{u}^+ = 0$$

## Sensitivity analysis perturbation of the baseflow field



• Unperturbed problem:

$$\sigma \mathbf{u} + \mathbf{u} \cdot \nabla \mathbf{U}_b + \mathbf{U}_b \cdot \nabla \mathbf{u} + \nabla p - \frac{1}{Re} \nabla^2 \mathbf{u} = \mathbf{0}$$
$$\nabla \cdot \mathbf{u} = 0$$



### Sensitivity analysis perturbation of the baseflow field



• Perturbed problem:

$$\tilde{\sigma}\mathbf{u} + \tilde{\mathbf{u}} \cdot \nabla \tilde{\mathbf{U}}_b + \tilde{\mathbf{U}}_b \cdot \nabla \tilde{\mathbf{u}} + \tilde{\nabla} p - \frac{1}{Re} \nabla^2 \tilde{\mathbf{u}} = \mathbf{0}$$
$$\nabla \cdot \tilde{\mathbf{u}} = 0$$



### Sensitivity analysis perturbation of the baseflow field



• Perturbed problem:

$$\tilde{\sigma}\mathbf{u} + \tilde{\mathbf{u}} \cdot \nabla \tilde{\mathbf{U}}_b + \tilde{\mathbf{U}}_b \cdot \nabla \tilde{\mathbf{u}} + \tilde{\nabla p} - \frac{1}{Re} \nabla^2 \tilde{\mathbf{u}} = \mathbf{0}$$
$$\nabla \cdot \tilde{\mathbf{u}} = 0$$

- Result of the sensitivity analysis (adjoint stab. equations involved) for mode  $(\sigma, \mathbf{u}, p)$ 

$$\delta \sigma = \frac{(M^+, \delta \mathbf{U}_b)}{(\mathbf{\hat{u}}^+, \mathbf{\hat{u}})}$$
$$M^+ = \mathbf{\hat{u}}^* \cdot \nabla \mathbf{\hat{u}}^+ - \nabla \mathbf{\hat{u}}^* \cdot \mathbf{\hat{u}}^+$$

# Sensitivity analysis <u>perturbation of the base-flow equations</u>

• Unperturbed problem:

$$\mathbf{U}_b \cdot \nabla \mathbf{U}_b + \nabla P_b - \frac{1}{Re} \nabla^2 \mathbf{U}_b = \mathbf{0}$$
$$\nabla \cdot \mathbf{U}_b = 0$$
$$\sigma \mathbf{u} + \mathcal{L}(\mathbf{U}_b, P_b) \mathbf{u} = 0$$



# Sensitivity analysis perturbation of the base-flow equations

• Perturbed problem:

$$\begin{split} \tilde{\mathbf{U}}_{b} \cdot \nabla \tilde{\mathbf{U}}_{b} + \nabla \tilde{P}_{b} - \frac{1}{Re} \nabla^{2} \tilde{\mathbf{U}}_{b} = \delta \mathbf{F}(\tilde{\mathbf{U}}_{b}, \tilde{P}_{b}) \\ \nabla \cdot \tilde{\mathbf{U}}_{b} = 0 \\ \tilde{\sigma} \tilde{\mathbf{u}} + \mathcal{L}(\tilde{\mathbf{U}}_{b}, \tilde{P}_{b}) = 0 \end{split}$$



# Sensitivity analysis <u>perturbation of the base-flow equations</u>

• Perturbed problem:

- Result of the sensitivity analysis for mode  $(\sigma, \mathbf{u}, p)$ 

$$\delta \sigma = \frac{\langle \mathbf{U}_{\mathbf{b}}^{+}, \delta \mathbf{F} \rangle}{\langle \hat{\mathbf{u}}^{+}, \hat{\mathbf{u}} \rangle}$$

Forced adjoint problem associated to the baseflow equations

$$\nabla \mathbf{U}_{\mathbf{b}} \cdot \mathbf{U}_{b}^{+} - \mathbf{U}_{\mathbf{b}} \cdot \nabla \mathbf{U}_{\mathbf{b}}^{+} + \nabla P_{b}^{+} - \frac{1}{Re} \nabla^{2} \mathbf{U}_{\mathbf{b}}^{+} = \mathbf{\hat{u}}^{*} \cdot \nabla \mathbf{\hat{u}}^{+} - \nabla \mathbf{\hat{u}}^{*} \cdot \mathbf{\hat{u}}^{+}$$
$$\nabla \cdot \mathbf{U}_{\mathbf{b}}^{+} = 0$$

### Sensitivity analysis discrete vs continuous approaches

**UNIVERSITÀ DI PISA** 



Discrete approach: starting point is the semi-discretized (in space) Navier-Stokes equations (quadratic ODE): the stability problem and related adjoint problems are derived directly for the discrete ODE system.

The two approaches are in sense parallel.....



• Semi-discretized (in space) NS equations; resulting ODE:

$$B_{ij}\frac{d\,u_j^c}{dt} + N_{ijk}\,u_j^c\,u_k^c + L_{ij}u_j^c = 0$$

- Discrete eigenvalue problem:  $N_{ijk} U_j U_k + L_{ij} U_j = 0$   $\underbrace{(N_{i,j,k} U_k + N_{i,k,j} U_k + L_{ij})}_{L_{ij}^C(\mathbf{U})} u_j + \sigma B_{ij} u_j = 0$ Eigenvalue probl.
  - Perturbed discrete eigenvalue problem:

$$N_{ijk} \tilde{U}_j \tilde{U}_k + L_{ij} \tilde{U}_j = P_{ij} \tilde{U}_j$$
Perturbation
$$\tilde{L}_{ij}^C (\mathbf{U}) \tilde{u}_j + \tilde{\sigma} B_{ij} \tilde{u}_j = Q_{ij} \tilde{u}_j$$



- Objective of the analysis: find the perturbation of the eigenvalue as a function of generic perturbations delta P, delta Q around the unperturbed system (P=Q=0)
- Straighforward method: lagrangian formulation (index repetition implies summation)





Variations of J with respect to the single variables is set to 0:

$$\begin{split} \frac{\delta J}{\delta z_i} \delta z_i &= 0 \to N_{ijk} U_j U_k + L_{ij} U_j - P_{ij} U_j = 0 & \text{Original baseflow equations} \\ \frac{\delta J}{\delta y_i} \delta y_i &= 0 \to L_{ij}^C, u_j + \sigma B_{ij} u_j - Q_{ij} u_j = 0 & \text{Original eigenvalue problem} \\ \frac{\delta J}{\delta u_j} \delta u_j &= 0 \to y_i^* L_{ij}^C + y_i^* \sigma B_{ij} = 0 & \text{Adjoint linear eigenvalue problem} \\ \frac{\delta J}{\delta U} \delta U_j &= 0 \to z_i^* (N_{ijk} U_k + N_{ikj} U_k + L_{ij} - P_{ij}) + y_i^* (N_{ikj} + N_{ijk} + L_{ij}) u_j = 0 & \text{Adjoint forced baseflow problem} \\ \frac{\delta J}{\delta \sigma} \delta \sigma &= 0 \to y_i^* B_{ij} u_j = 1 & \text{Normalization cond.} & \text{Adjoint forced baseflow problem} \\ \frac{\delta J}{\delta \sigma} \delta Q &= \frac{\delta \sigma}{\delta P} \delta P = z_i^* \delta P_{ij} U_j & \text{Eigenvalue variation with respect to a generic matrix P} \\ \frac{\delta J}{\delta Q} \delta Q &= \frac{\delta \sigma}{\delta Q} \delta Q = y_i^* \delta Q_{ij} u_j & \text{Eigenvalue variation with respect to a generic matrix Q} \\ \text{June 04 2013, Poitiers} \end{split}$$



<b>Discrete formulation</b>	<b>Continuous formulation</b>
$N_{ijk} U_j U_k + L_{ij} U_j = 0$	$egin{aligned} \mathbf{U}_b \cdot  abla \mathbf{U}_b +  abla P_b - rac{1}{Re}  abla^2 \mathbf{U}_b = 0 \  abla \cdot \mathbf{U}_b = 0 \end{aligned}$
$L_{ij}^C, u_j + \sigma B_{ij}u_j - Q_{ij}u_j = 0$	$\sigma \mathbf{u} + \mathbf{u} \cdot \nabla \mathbf{U}_b + \mathbf{U}_b \cdot \nabla \mathbf{u} + \nabla p - \frac{1}{Re} \nabla^2 \mathbf{u} = 0$ $\nabla \cdot \mathbf{u} = 0$
$y_i^* L_{ij}^C + y_i^* \sigma B_{ij} = 0$	$\sigma^* \mathbf{u}^+ + \nabla \mathbf{U}_b \cdot \mathbf{u}^+ - \mathbf{U}_b \cdot \nabla \mathbf{u}^+ + \nabla p^+ - \frac{1}{Re} \nabla^2 \mathbf{u}^+ = 0$ $\nabla \cdot \mathbf{u}^+ = 0$
$z_i^* (N_{ijk} U_k + N_{ikj} U_k + L_{ij} - P_{ij}) = = -y_i^* (N_{ikj} + N_{ijk} + L_{ij}) u_j = 0$	$\nabla \mathbf{U}_{\mathbf{b}} \cdot \mathbf{U}_{b}^{+} - \mathbf{U}_{\mathbf{b}} \cdot \nabla \mathbf{U}_{\mathbf{b}}^{+} + \nabla P_{b}^{+} - \frac{1}{Re} \nabla^{2} \mathbf{U}_{\mathbf{b}}^{+} = \hat{\mathbf{u}}^{*} \cdot \nabla \hat{\mathbf{u}}^{+} - \nabla \hat{\mathbf{u}}^{*} \cdot \hat{\mathbf{u}}^{+}$ $\nabla \cdot \mathbf{U}_{\mathbf{b}}^{+} = 0$
$\frac{\delta\sigma}{\delta P}\delta P = z_i^*\delta P_{ij}U_j$	$\delta \sigma = rac{({f U_b}^+, \delta {f F})}{({f \hat u}^+, {f \hat u})}$
$\frac{\delta\sigma}{\delta Q}\delta Q = y_i^*\delta Q_{ij}u_j$	$\delta \sigma = \frac{\langle \mathbf{u}^+, \delta \mathbf{H}(\mathbf{u}, \mathbf{p}) \rangle}{\langle \mathbf{u}^+, \mathbf{u} \rangle}$

June 04 2013, Poitiers

# Applications to flow analysis and control



UNIVERSITÀ DI PISA

Perturbation of linearized equation: Computational Prop. Feedback control of vortex shedding S. Camarri and A. Iollo, PoF 22(094102), 2010 Perturbation of the base-flow equations Passive control of a pitchfork bifurcation by a control  $\geq$ cylinder in the flow A. Fani, S. Camarri, and M. V. Salvetti, PoF 24(084102), 2012. 6 Sensitivity analysis at high Reynolds number: 0.8 Application to PIV data past a porous cylinder S. Camarri, B. E. G. Fallenius, and J. H. M. Fransson, JFM 715, 2013. -0.5 -0.2 -1.01.5 2.0 2.5 Sensitivity analysis and control maps for fully 3D configs: Application to a fully 3D T-Mixer A. Fani, S. Camarri, and M. V. Salvetti, accepted, PoF 2013

Perturbation of linearized equation: Prop. Feedback control of vortex shedding **S. Camarri and A. Iollo, PoF 22(094102), 2010** 

### Applications to flow analysis and control

Perturbation of linearized equation for the design of a proportional feedback control of vortex shedding past a bluff body

# Flow control by means of a perturbation of the linearized disturbance equations

- Typical case when the applied control leaves the mean flow unaltered and the control acts only on the linearized equations
- Example: a feedback control based only on the perturbation field to control the vortex shedding instability past a cylinder:
  - Proof of concept for the use of sensitivity analysis for flow control
  - Realizable control: a few velocity probes, surface jets as actuators, simple proportional feedback control
  - Objective of control: to make the steady flow linearly stable
  - Proposal of an iterative strategy because the control based solely on the sensitivity analysis of uncontrolled flow can be misleading (action of control out of the linear range).



Flow control by menas of a perturbation of the linearized disturbance equations

• Flow configuration (incompressible 2D flow):









UNIVERSITÀ DI PISA

• Flow configuration (incompressible 2D flow):



Jet width: 0.16 L Comp. domain dims:  $-12.5L \le x \le 20.5L$  $-4L \le x \le 4L$ GR1 540X330 (~5 $\cdot$ 10<sup>5</sup> dof) GR2 810X494 (~**10<sup>6</sup> dof**)

- Reference quantities: Uc, L
- Critical Reynolds number for primary instability:  $Re_{cr} \simeq 59$
- Objective of control: linearly stabilize the steady unstable solution for Re>59



Flow control by menas of a perturbation of the linearized disturbance equations

• Flow configuration (incompressible 2D flow):



- Boundary conditions on the jet surface S<sub>j</sub> :proportional feedback from a set of ideal velocity probes.
- Only the difference between the measured flow and the steady unstable field is fed back to the actuators: **the steady unstable flow remains a solution of the controlled flow**



• This control perturbs only the linearized stability equations:

$$L_{ij}^C, u_j + \sigma B_{ij}u_j - Q_{ij}u_j = 0$$

• The control matrix Q at discrete level can be represented as follows:

$$\mathbf{Q} = -\sum_{s=1}^{N_s} K_s \mathbf{C}(x_s, y_s, \theta_s)$$





• This control perturbs only the linearized stability equations:

$$L_{ij}^C, u_j + \sigma B_{ij}u_j - Q_{ij}u_j = 0$$

• The control matrix Q at discrete level can be represented as follows:





• This control perturbs only the linearized stability equations:

$$L_{ij}^C, u_j + \sigma B_{ij}u_j - Q_{ij}u_j = 0$$

• The control matrix Q at discrete level can be represented as follows:

$$\mathbf{Q} = -\sum_{s=1}^{N_s} K_s \mathbf{C}(x_s, y_s, \theta_s)$$

• The effect of perturbation is computed by the adjoint stab. problem:

$$y_i^* L_{ij}^C + y_i^* \sigma B_{ij} = 0$$
$$\delta \sigma = y_i^* \delta Q_{ij} u_j$$

• To design the control we need to know the perturbation matrix Q as a function of the control parameters (feedback coeff and position of probes)

$$\delta \mathbf{Q} = \sum_{s=1}^{N_s} \left\{ \left[ \mathbf{C}(x_s, y_s, \theta_s) \right] \, \delta K_s + \left[ K_s \frac{\partial \mathbf{C}}{\partial x_s}(x_s, y_s, \theta_s) \right] \, \delta x_s + \left[ K_s \frac{\partial \mathbf{C}}{\partial y_s}(x_s, y_s, \theta_s) \right] \, \delta y_s + \left[ K_s \frac{\partial \mathbf{C}}{\partial \theta_s}(x_s, y_s, \theta_s) \right] \, \delta \theta_s \right\}$$



### Design of the control

The adjoint analysis allows to predict the shift of the eigenvalue of the discrete system as the control parameters (feedback coefficients, sensor positions, measured veloc. component) are perturbed.







We can design the control strategy iteratively, by driving the unstable eigenvalues in the stable region of the complex plane



### Design of the control



• Stable eigenvalues are perturbed also by the control and may eventually become unstable

• Adopted strategy to control the spectrum: define a function of the spectrum of the controlled system such that the minimum is reached when the system is stabilized. The previous analysis allows the computation of the gradient of the function.

• The control is designed by minimizing this function f, which depends on the control parameters



#### Design of the control





Iterative process: to bypass the fact that the sensitivity analysis gives results that are accurate only for a small perturbation of the control parameters (linearized analysis).

June 04 2013, Poitiers




June 04 2013, Poitiers

### Results



Examples of controls using one velocity sensor placed in the wake on the symmetry line, measuring the vertical component of velocity.

Control parameters: x<sub>s</sub>, k<sub>s</sub>



Results: sensitivity maps with respect to the uncontrolled state (contraint: y<sub>s</sub>=0, vert. veloc)





Results: examples of control at Re=90 (constraint: one vert.veloc. sensor with  $y_s = 0^{T}$ ) ERSITA DI PISA



June 04 2013, Poitiers







EPFL Lausanne, 16 June 2011

June 04 2013, Poitiers

## Attraction basin of the stabilized system (a-posteriori tests)



The design guarantees that, when finished with success, the steady solution is linearly stable. The basin of attraction of this stable point for the system needs to be explored a-posteriori.



Perturbation of the base-flow equations Passive control of a pitchfork bifurcation by a control cylinder in the flow



A. Fani, S. Camarri, and M. V. Salvetti, PoF 24(084102), 2012.

### Applications to flow analysis and control

#### Passive control of the pitchfork instability in a symmetric plane channel with a sudden expansion



Incompressible flow in a 2-D plane channel with ER=D/d=3



# Symmetric plane channel with a sudden expansion





### I: Stability analysis





• A small control cylinder of diameter  $d^*$ , introduced in the channel at the position  $(x_0, y_0)$ 



- A small control cylinder of diameter d\*, introduced in the channel at the position (x<sub>0</sub>, y<sub>0</sub>)
- 2 The cylinder is modeled by the force it exerts on the flow



- A small control cylinder of diameter d\*, introduced in the channel at the position (x<sub>0</sub>, y<sub>0</sub>)
- On the cylinder is modeled by the force it exerts on the flow
- Linearized drag force:

$$F \approx -\alpha \left[ \|U_b\| (U_b + \hat{u}) + \left(\frac{U_b}{\|U_b\|} \cdot \hat{u}\right) U_b \right] \delta(x - x_0, y - y_0)$$



- A small control cylinder of diameter d\*, introduced in the channel at the position (x<sub>0</sub>, y<sub>0</sub>)
- On the cylinder is modeled by the force it exerts on the flow
- Inearized drag force:

$$F \approx -\alpha \left[ \|U_b\| (U_b + \hat{u}) + \left(\frac{U_b}{\|U_b\|} \cdot \hat{u}\right) U_b \right] \delta(x - x_0, y - y_0)$$

 α = α(d\*) The force amplitude is a function of the cylinder diameter.



- A small control cylinder of diameter d\*, introduced in the channel at the position (x<sub>0</sub>, y<sub>0</sub>)
- On the cylinder is modeled by the force it exerts on the flow
- Linearized drag force:

$$F \approx -\alpha \left[ \|U_b\| (U_b + \hat{u}) + \left(\frac{U_b}{\|U_b\|} \cdot \hat{u}\right) U_b \right] \delta(x - x_0, y - y_0)$$

 α = α(d\*) The force amplitude is a function of the cylinder diameter.

#### Eigenvalue variation caused by the cylinder

The drag is a function of both base flow and perturbation.  $\delta\sigma = \delta\sigma(U_b, \hat{u})$ 

We can estimate the effect of both the two different contributions to the force on the instability using the previous analysis.



the cylinder

diameter  $d^*$ 

### II: passive control strategy

We can define a sensitivity function **S**, which gives the eigenvalue variation for each position of the cylinder  $(x_0, y_0)$ :  $\alpha$  is a function of

$$\delta\sigma = \alpha S(x_0, y_0)$$



June 04 2013, Poitiers



Cylinder introduced impulsively, starting from the asymmetric state

 $d^* = 0.02 x_0 = 0.15 y_0 = 0$ 





Università di Pisa

Cylinder introduced impulsively, starting from the asymmetric state

 $d^* = 0.02 x_0 = 0.15 y_0 = 0$ 





UNIVERSITÀ DI PISA



Re=100 
$$d^* = 0.02$$

t=25





June 04 2013, Poitiers





June 04 2013, Poitiers



Asymmetric deviation: the streamline on the centerline is deviated towards the part where the separation is smaller

Re=100  $d^* = 0.02$ 











Re=100 
$$d^* = 0.02$$

- Two effects:
- Convection of vorticity from the smaller channels
- 2. Alteration of the streamlines and production of vorticity at the channel walls.





We try to isolate one of the two effect to understand their role in the control



The initial state is the controlled symmetric solution

Ad hoc simulation where the production of new vorticity from the walls in the larger channel is annihilated :

- Navier Stokes equations in terms of disturbance
- Slip condition for the perturbation



We try to isolate one of the two effect to understand their role in the control



Ad hoc simulation where the production of new vorticity from the walls in the larger channel is annihilated :

- Navier Stokes equations in terms of disturbance
- Slip condition for the perturbation

Modifications in terms of convection of vorticity from the smaller channel is not sufficient to control the instability



Eigenvalue drift evaluated with sensitivity analysis is exact only for infinitesimal perturbations.

 $\overline{\lambda} = \lambda_0 + \alpha S(x_0, y_0)$  where  $\overline{\lambda}$  is the growth rate predicted by the sensitivity analysis and  $\lambda_0$  is the growth rate for the uncontrolled flow.



Eigenvalue drift evaluated with sensitivity analysis is exact only for infinitesimal perturbations.

 $\overline{\lambda} = \lambda_0 + \alpha S(x_0, y_0)$  where  $\overline{\lambda}$  is the growth rate predicted by the sensitivity analysis and  $\lambda_0$  is the growth rate for the uncontrolled flow.

We can retreive some non-linear effects due to a finite amplitude of the forcing with the following procedure:

• Application of the linearized drag force at a fixed position  $(x_0 = 0.15, y_0 = 0)$ 



Eigenvalue drift evaluated with sensitivity analysis is exact only for infinitesimal perturbations.

 $\overline{\lambda} = \lambda_0 + \alpha S(x_0, y_0)$  where  $\overline{\lambda}$  is the growth rate predicted by the sensitivity analysis and  $\lambda_0$  is the growth rate for the uncontrolled flow.

We can retreive some non-linear effects due to a finite amplitude of the forcing with the following procedure:

- Application of the linearized drag force at a fixed position  $(x_0 = 0.15, y_0 = 0)$
- 2 Computation of the forced baseflow for different amplitudes ( $\alpha$ ):  $U'_b \cdot \nabla U'_b + \nabla P'_b - \frac{1}{Re} \nabla^2 U'_b = \delta F$  $\nabla \cdot U'_b = 0$



Eigenvalue drift evaluated with sensitivity analysis is exact only for infinitesimal perturbations.

 $\overline{\lambda} = \lambda_0 + \alpha S(x_0, y_0)$  where  $\overline{\lambda}$  is the growth rate predicted by the sensitivity analysis and  $\lambda_0$  is the growth rate for the uncontrolled flow.

We can retreive some non-linear effects due to a finite amplitude of the forcing with the following procedure:

- Application of the linearized drag force at a fixed position  $(x_0 = 0.15, y_0 = 0)$
- Ocmputation of the forced baseflow for different amplitudes ( $\alpha$ ):  $U'_b \cdot \nabla U'_b + \nabla P'_b \frac{1}{Re} \nabla^2 U'_b = \delta F$   $\nabla \cdot U'_b = 0$

3 Linear stability analysis on the forced baseflow  $U'_{b} \rightarrow \lambda'_{b}$ 





Sensitivity analysis at high Reynolds number: Application to PIV data past a porous cylinder

S. Camarri, B. E. G. Fallenius, and J. H. M. Fransson, JFM 715, 2013.

#### Applications to flow analysis and control

1.0

0.5

-0.5

-1.0

0.5

1.5

2.0

25

0.8 ITÀ DI PISA

n 6

0.4

0.2

-0.2

#### Stability and sensitivity analysis of experimental flow fields measured past a porous cylinder

Funding by C.M. Lerici Foundation is gratefully acknowledged



## Configuration: flow around a porous cylinder (uniform transpiration)





#### **Experimental PIV database**





### Stability analysis of mean flow fields

- For bluff-body flows correct prediction of the Strouhal number of vortex shedding, associated to a nealy marginally stable mode
- Shown by DNS up to Re=600 (*Leontini et al., JFM 2010*)
- Inspiring physical interpretation in terms of baseflow modifications in a ROM framework (*Noack et al., JFM 2003*)
- Models to include Reynolds stresses (e.g. *Reynolds & Hussain, JFM 1972; Kitsios et al., JFM 2010*) here neglected.

### Objectives



- Verify whether or not global stability analysis, when applied to the experimental mean flow fields at Re=3700, still predicts the vortex shedding (VS) frequency with a sufficient accuracy to highlight the effect of Γ
  - Difficulties: noisy data, low spatial resolution, small measuring window
- Use the results of the stability analysis to extract information about the large scale vortical structures from available database
- 3. Provide a strategy to apply sensitivity analysis for flow control using mean flow fields (similar work based on RANS simulations in *Meliga et al. PoF 2012*).



#### Stability analysis - procedure







#### Stability analysis - procedure

3) Computation of the associated **adjoint mode** (Γ=-2.57)






## Stability analysis - procedure

4) Localization of the overlapping between direct and adjoint mode: core of the instability, i.e. region of the baseflow which mostly affect the estimation of the global mode (*Giannetti & Luchini, JFM*)





## Stability analysis - procedure





75

## Stability analysis - procedure







## Stability analysis - results

Γ	Resolution	Stabilized	Strouhal N.	Experimental St	Error (%)
-6	120X120	Y	stable	stable	_
-5	120X120	Y	0.290	—	_
-3.86	120X120	Y	0.266	—	—
-3.21	120X120	Y	0.255	—	_
-2.57	120X120	Y	0.285	0.283	0.7%
-1.93	120X120	Y	0.267	0.241	9.7%
$-1.37^{*}$	120X120	Y	0.308	0.216	30.0%
0	120X120	Y	0.232	0.2	13.8%
+0.68	120X120	Y	0.218	0.190	12.8%
+1.93	120X120	Y	0.193	0.188	2.6%
$+2.57^{*}$	120X120	Y	0.1919	0.176	8.31%



## Stability analysis - results

Γ	Resolution	Stabilized	Strouhal N.	Experimental St	Error $(\%)$
-6	120X120	Y	stable	stable	_
-5	120X120	Υ	0.290	_	_
-3.86	120X120	Y	0.266	_	_
-3.21	120X120	Y	0.255	-	—
-2.57	120X120	Y	0.285	0.283	0.7%
-1.93	120X120	Υ	0.267	0.241	9.7%
$-1.37^{*}$	120X120	Y	0.308	0.216	30.0%
0	120X120	Y	0.232	0.2	13.8%
+0.68	120X120	Υ	0.218	0.190	12.8%
+1.93	120X120	Y	0.193	0.188	2.6%
$+2.57^{*}$	120X120	Y	0.1919	0.176	8.31%

• Accuracy on the value of St within 14 %



### Stability analysis - results



- Accuracy on the value of St within 14 %
- Variations of St vs Γ in agreement with experiments



- Accuracy on the value of St within 14 %
- Variations of St vs  $\Gamma$  in agreement with experiments
- Errors when the instability core approaches the boundaries of the measurement window

## Sensitivity analysis perturbation of the baseflow field



• Perturbed problem:

$$\tilde{\sigma}\mathbf{u} + \tilde{\mathbf{u}} \cdot \nabla \tilde{\mathbf{U}}_b + \tilde{\mathbf{U}}_b \cdot \nabla \tilde{\mathbf{u}} + \tilde{\nabla} p - \frac{1}{Re} \nabla^2 \tilde{\mathbf{u}} = \mathbf{0}$$
$$\nabla \cdot \tilde{\mathbf{u}} = 0$$

• Result of the sensitivity analysis (adjoint stab. equations involved):

$$\delta \sigma = \frac{(M^+, \delta \mathbf{U}_b)}{(\hat{\mathbf{u}}^+, \hat{\mathbf{u}})}$$
$$M^+ = \hat{\mathbf{u}}^* \cdot \nabla \hat{\mathbf{u}}^+ - \nabla \hat{\mathbf{u}}^* \cdot \hat{\mathbf{u}}^+$$



50

# Sensitivity of mode frequency a generic baseflow modification



Sensitivity to a generic perturbation of the baseflow horizontal velocity







# Sensitivity of mode frequency a generic baseflow modification





60

40

20

0

-20

-40

-60

## Sensitivity of mode frequency a generic baseflow modification

Case at Re=50 computed on the steady Sensitivity to a generic perturbation of unstable solution of NS eqs. the baseflow horizontal velocity 0.5 3 0.4 2 0.3 0.2 0.5 0.1 0 C -0.1 -0.5 -1 -0.2 -0.3 -1 -0.4 0.5 2.5 0 1.5 2 -0.5

З



# Sensitivity of mode frequency a generic baseflow modification





# Focus on control: sensitivity to a generic baseflow modification

• Verification of the sensitivity maps: control parameter Γ





# Focus on control: sensitivity to a generic baseflow modification

- Example of application:
  - control parameter: transpiration Γ
  - From PIV measurements the vatiation of baseflow is known
  - We use the map derived for  $\Gamma$ =-1.93 to estimate variations of Strouhal number thanks to the result of sensitivity analysis:

Γ	Estimated Strouhal N.	Computed Strouhal N.	Difference $(\%)$
-5.0	0.333	0.290	14.8
-3.86	0.261	0.266	-1.9
-3.21	0.252	0.255	-1.2
-2.57	0.282	0.285	-1.0
0	0.223	0.232	-3.9
0.68	0.210	0.218	-3.7
1.93	0.161	0.193	-16.6



# Example of flow analysis based on stability results: phase alignment of snapshots

- Use of global modes to extraction of information on the large scale wake vortices
  - Global modes (stability analysis) are used to estimate the phase of each snapshot with respect to VS
  - Snapshots are aligned with respect to the VS phase
  - Phase average carried out on the ordered database





## Phase alignment of snapshots

- Extraction of information on the large scale wake vortices
  - Global modes (stability analysis) are used to estimate the phase of each snapshot with respect to VS
  - Snapshots are aligned with respect to the VS phase
  - Phase average carried out on the ordered database
- Phase estimation

$$\mathbf{u}_{vs}(x, y, t) = \mathbf{U}_{\mathbf{m}} + \frac{A}{2}(\exp(\mathrm{i}\omega t) \ \hat{\mathbf{u}}(x, y) + \exp(-\mathrm{i}\omega t) \ \hat{\mathbf{u}}^*(x, y))$$

First strategy (only global mode nec.)

$$\mathbf{r} = \hat{\mathbf{u}} - \frac{\langle \hat{\mathbf{u}}, \hat{\mathbf{u}}^* \rangle}{\langle \hat{\mathbf{u}}^*, \hat{\mathbf{u}}^* \rangle} \hat{\mathbf{u}}^*$$
$$\Phi_T = \text{Phase}(\langle \mathbf{r}, \mathbf{u}(x, y, t) - \mathbf{U}_{\mathbf{m}} \rangle)$$

Second strategy (adjoint based)

$$\frac{A}{2}\exp(\mathrm{i}\Phi_T) = \left\langle \hat{\mathbf{u}}^+, \mathbf{u}(x, y, t) - \mathbf{U}_{\mathbf{m}} \right\rangle$$



## Phase alignment of snapshots Tipical example (Γ=-2.57)



## Phase alignment of snapshots Tipical example (Γ=-2.57)







## Phase alignment of snapshots Blowing case (Γ=0)





Phase averaged vorticity field

# • Far and close windows: two different experimental runs

 Sincronization is automatic because a single global mode is used to identify the phase



## On-going developments

- Development passive wake controls using only experimental mean flow fields and final implementation in experiments for a thick plate (KTH Mechanics)
- Use of adjoint methods for data reconstruction problems in support to experiments on (KTH Mechanics):
  - Separated wakes
  - Controlled boundary layers
- Applications to fully 3D mixers for microfluidics applications:
  - Flow analysis
  - Control applications oriented to mixing enhancement
- Inclusion of Reynolds stresses as closure for local stability analysis of experimental flow fields past wind turbines (EPFL Lausanne)
- Application to free falling bluff bodies (Univ. Bordeaux)

Sensitivity analysis and control maps for fully 3D configs: Application to a fully 3D T-Mixer

A. Fani, S. Camarri, and M. V. Salvetti, accepted, PoF 2013

## Applications to flow analysis and control

ΔZ

ITÀ DI PISA

# Sensitivity analysis of the engulfment instability in a fully 3D micro T-Mixer

### Micro T-mixer



T-mixer are very common devices in microfluidics, also used as junction elements in complex micro-systems  $\longrightarrow$  laminar regime (low Reynolds numbers)





#### Flow regimes as a function of Reynolds number

- Segregated flow in the outflow pipe Stratified flow •
- Vortex regimeEngulfment

Re



#### Flow regimes as a function of Reynolds number

- Stratified flow
- Vortex regime
- Engulfment

Secondary flow in the outcoming pipe: double pair of counter rotating vortices





Re



#### Flow regimes as a function of Reynolds number

- Stratified flow
- Vortex regime
- Engulfment

Re





Stationary and organized pattern of vortical structures, which improve mixing in the outflow pipe.



#### Flow regimes as a function of Reynolds number





High sensitivity of the engulfment to inflow boundary conditions (Galletti et al.(2012)).

If the flow at the confluence is not fully developed engulfment occurs at larger Reynolds numbers.



- High sensitivity to inflow conditions, as well as to other parameters, have been shown in the literature through numerical simulations and experiments
- No previous stability analysis of this configuration
- In general, stability analysis of fully 3D flows are rare

## Objectives

- Aim of the present work: to carry out a systematic investigation by means of linear instability and sensitivity analyses to explain high sensitivity to inflow conditions
- Development of tools for 3D configurations
- Investigation of the flow also by DNS



## Flow configuration

Incompressible flow in a three dimensional T-mixer



- $W_0/H = 1.5$  and  $W_0/W_i = 2$  (same geometry as in Galletti et al. (2012))
- Reference length:  $D_h = \frac{2W_O H}{W_O + H}$  (hydraulic diameter)
- Reference velocity: U<sub>m</sub> bulk velocity of the inlet flow (fully developed inflow condition)



#### Re=140: vortex regime



Two counter-rotating vortices in the separated region at the confluence

Two pair of vortices observed in the mixing channel

Vortex identified by  $\lambda_2$  criteria



#### Re=140: vortex regime



Normal vorticity component





#### Re=140: vortex regime





#### Re=160: engulfment







June 04 2013, Poitiers









June 04 2013, Poitiers





Only two pair of vortices last after the first part of the mixer






### 3D stability analysis



## Re=140: just below the critical Reynolds number evaluated with DNS



### 3D stability analysis





- Well defined vortical structures
- Point symmetry with respect to the center of the cross section

Contours: normal to plane velocity Arrows: in-plane velocities



#### 3D stability analysis





The global mode is well correlated with the S-shaped engulfment pattern Contours: normal to plane velocity Arrows: in-plane velocities





The region where global and adjoint fields overlap is the intersection of inlet and outflowing pipes.

## Sensitivity to perturbation of the inlet velocity conditions



Following the approach of Marquet et al. (2008), we obtain:

$$\delta \sigma = \frac{\left\langle P_b^{+} \boldsymbol{n} + \boldsymbol{R} e^{-1} \boldsymbol{n}^T \nabla \boldsymbol{U}_b^{+} \delta \boldsymbol{U}_i \right\rangle_{\Gamma i}}{\left\langle \boldsymbol{u}^{+}, \boldsymbol{\hat{u}} \right\rangle}$$

**n**: normal unit vector to the boundary pointing outside the flow domain <\*,\*><sub>ri</sub> complex scalar product computed on the inlet surface

We observed that the flow is almost receptive only to a perturbation of the component of velocity normal to the inflow boundary:

$$\delta \sigma = \frac{\langle P_b^{+} \boldsymbol{n}, \delta \boldsymbol{U}_i \rangle_{\Gamma i}}{\langle \boldsymbol{u}^{+}, \widehat{\boldsymbol{u}} \rangle}$$

# Sensitivity to perturbation of the inlet velocity conditions

If we consider a perturbation of the form  $\delta U_i = U_i \delta(y, z)n$  we can write:

 $\delta \sigma = U_i S(y,z)$ 

**S** : sensitivity map of the eigenvalue with respect to a localized modification of the wall normal component of the inflow velocity, computed on the inlet surface.



- A decrease of the inflow velocity at a generic location of the inflow section always implies a negative δσ
- Influence of the location of the velocity perturbation on the stabilizing/destabilizing effect

June 04 2013, Poitiers



## Application of the sensitivity map

Velocity perturbation ( $U_i$ ) associated with a not fully developed inflow condition





## Application of the sensitivity map

Velocity perturbation ( $U_i$ ) associated with a not fully developed inflow condition



A stability analysis carried out on the base-flow with the non fully developed inflow conditions has confirmed the results.



#### Sensitivity maps for micro-jets at the T-mixer walls

The same sensitivity maps, computed on the mixer walls, can be used to evaluate the effect of micro-jets on the instability.



Suction ( $U_{jet} > 0$ ) has a destabilizing effect where S is positive valued



## On-going developments

- Development passive wake controls using only experimental mean flow fields and final implementation in experiments for a thick plate (KTH Mechanics)
- Use of adjoint methods for data reconstruction problems in support to experiments on (KTH Mechanics):
  - Separated wakes
  - Controlled boundary layers
- Applications to fully 3D mixers for microfluidics applications:
  - Flow analysis
  - Control applications oriented to mixing enhancement
- Inclusion of Reynolds stresses as closure for local stability analysis of experimental flow fields past wind turbines (EPFL Lausanne)
- Application to free falling bluff bodies (Univ. Bordeaux)